



Mooring forces

Classical approach to ensure smooth and safe F/E of lock:
“**hawser force** criterion”

- misleading term
- **hydrodynamic force** exerted by water on vessel should be below a given **threshold value**
- measure force on (centrally positioned) vessel in **scale model** (no mooring lines)



Mooring forces

Threshold values for **inland** navigation:

- popular value for quite sometime:
1‰ of displacement weight
- since 1980's (~neighbouring countries, mainly NL):

CEMT class	LxBxT	Filling (fixed bollards)	Filling (floating bollards), emptying
IV	80m x 9.45m x 2.8m	1.1 ‰	1.5 ‰
Va	135m x 11.4m x 3.5m	0.85 ‰	1.15 ‰
Vb	190m x 11.4m x 3.5m	0.75 ‰	0.75 ‰

Conservative...?
(in comparison to on site measurements)

Recreational vessels: 3 ‰

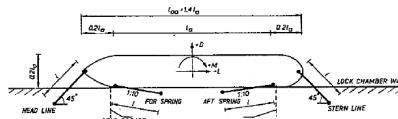
- relative approach: e.g. actual situation normative for situation after renovation of F/E system

Mooring forces

Threshold values for **sea-going** navigation:

- A. Vrijburcht (1977)

Conservative...?
(in comparison to on site measurements)



ship's size tons DW	rope's type	T_p/T_u	criteria in °/oo		
			long. force x	transv. force y	moment z
40,000	steel	0.15	0.24	0.14	0.075 0.050 0.025
40,000	steel	0.10	0.15	0.10	0.051 0.036 0.017
40,000	nylon	0.20	0.26	0.20	0.102 0.068 0.034
40,000	nylon	0.10	0.14	0.10	0.051 0.034 0.017
80,000	steel	0.15	0.21	0.12	0.066 0.044 0.022
80,000	steel	0.10	0.13	0.085	0.044 0.029 0.015
80,000	nylon	0.20	0.22	0.16	0.087 0.058 0.029
80,000	nylon	0.10	0.115	0.08	0.043 0.029 0.015
120,000	steel	0.15	0.19	0.11	0.060 0.040 0.020
120,000	steel	0.10	0.12	0.08	0.041 0.027 0.014
120,000	nylon	0.20	0.205	0.16	0.081 0.054 0.027
120,000	nylon	0.10	0.105	0.08	0.040 0.027 0.013

- relative approach: e.g. existing lock normative for
(± comparable) new lock



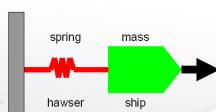
Mooring forces

Threshold values:

- (too?) often only longitudinal force component considered

- threshold = $\sum_{\text{mooring lines}} \frac{T_u \cos(\theta_i) \cos(\phi_i)}{f_s \cdot f_m}$

T_u = minimum tensile strength
 f_s = safety factor w.r.t. T_u
 f_m = dynamic magnification factor
 θ_i, ϕ_i = line orientation



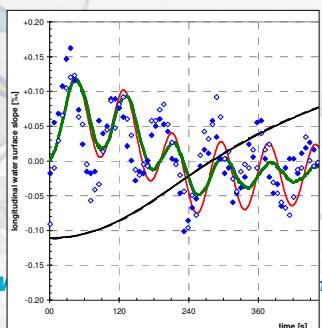
cf. H.-W. Partenscky (1986) ; A. Vrijburcht (1994) ; T. De Mulder (2007)

Mooring forces

Mathematical models for “lateral” F/E systems



- lock chamber flow based on Shallow Water Eqs. (1D or 2D)
- (pre)calculated timeseries of filling/emptying discharges
- vessel represented by means of artificial field of atm.pressure
- longitudinal water surface slope (bow-stern)



numerical predictions (**LOCKSIM**,**DELFT3D**)
and values measured on site

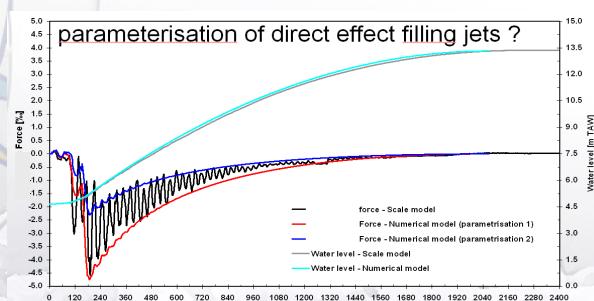
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Mooring forces

Mathematical model for “longitudinal” F/E systems



- LOCKFILL (NL ; developed since 1990's)
- similar programme developed in house at FHR
- intensive validation



- recreational navigation ?
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“Hawser Forces”

Numerical calculation / Physical model test on immobilized vessel

- (horizontal) force and moment components on ship due to Filling/Emptying = “**exciting forces**”
- to be applied to ship + moorings/control
- dynamic system
- forces in mooring system + lateral displacement

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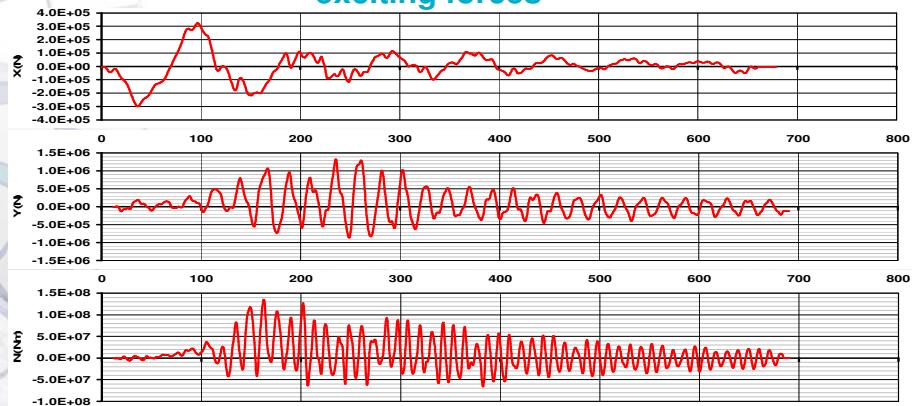
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“**exciting forces**”



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Dynamic system ?

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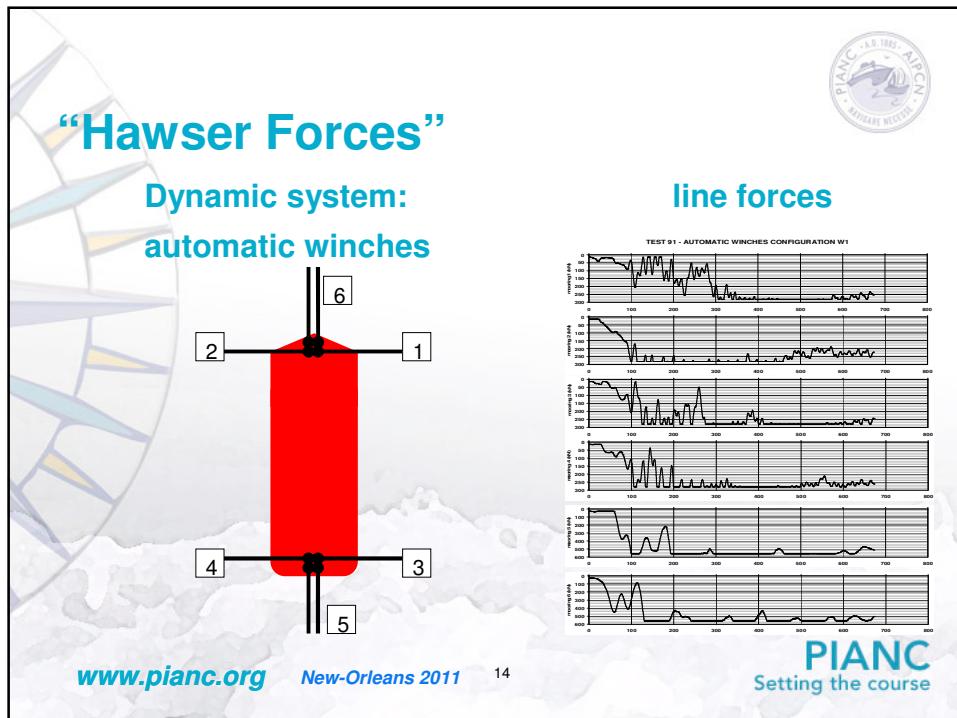
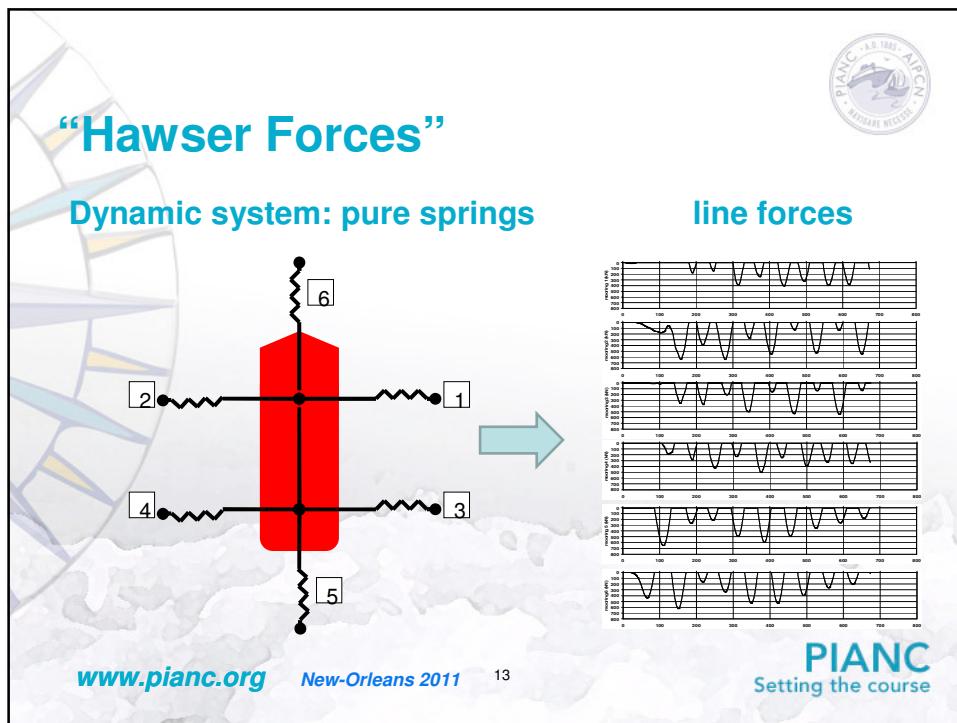
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Dynamic system: pure springs

motions

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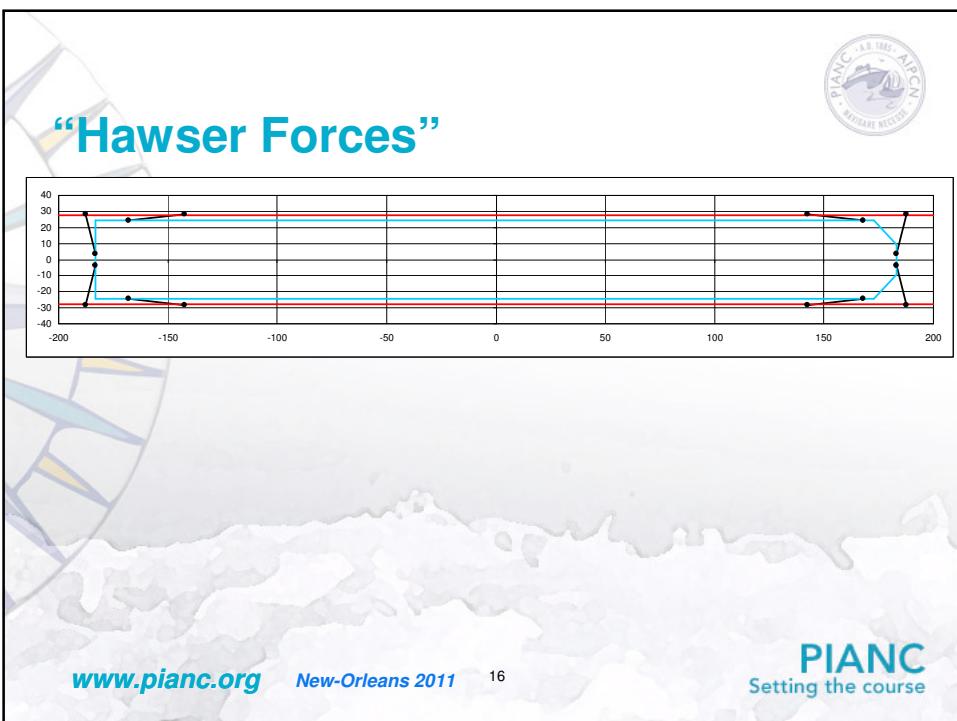
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Increase realism:

- Realistic mooring configuration
- Vertical motion!
- Winch control

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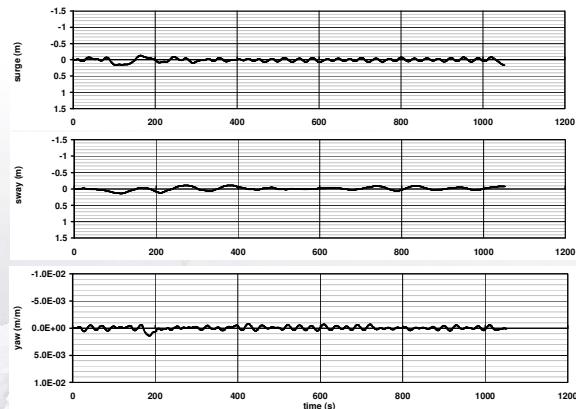
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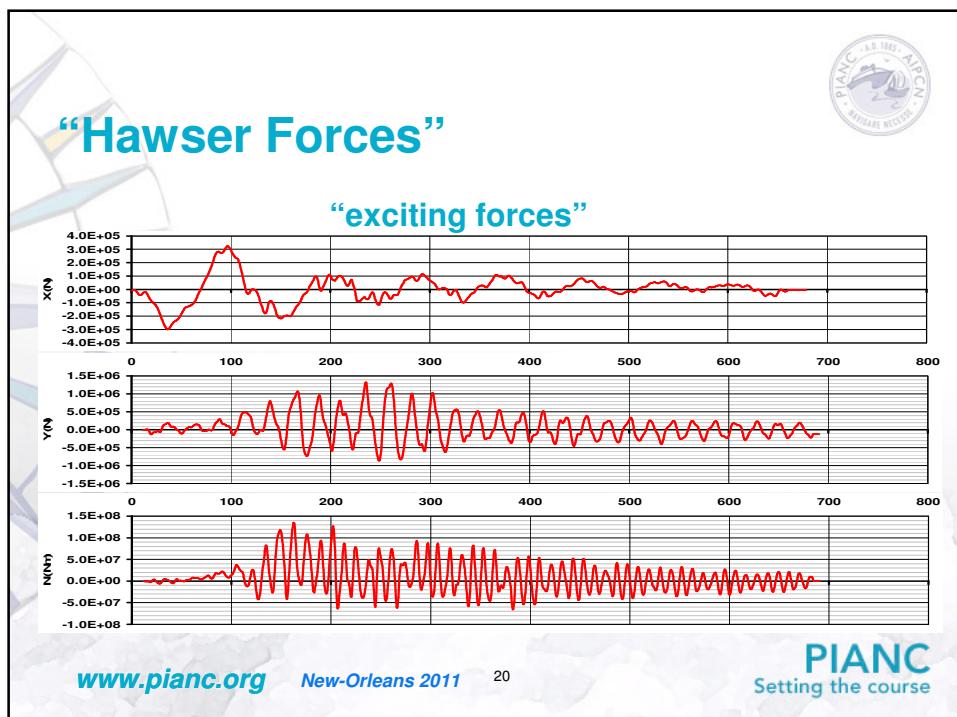
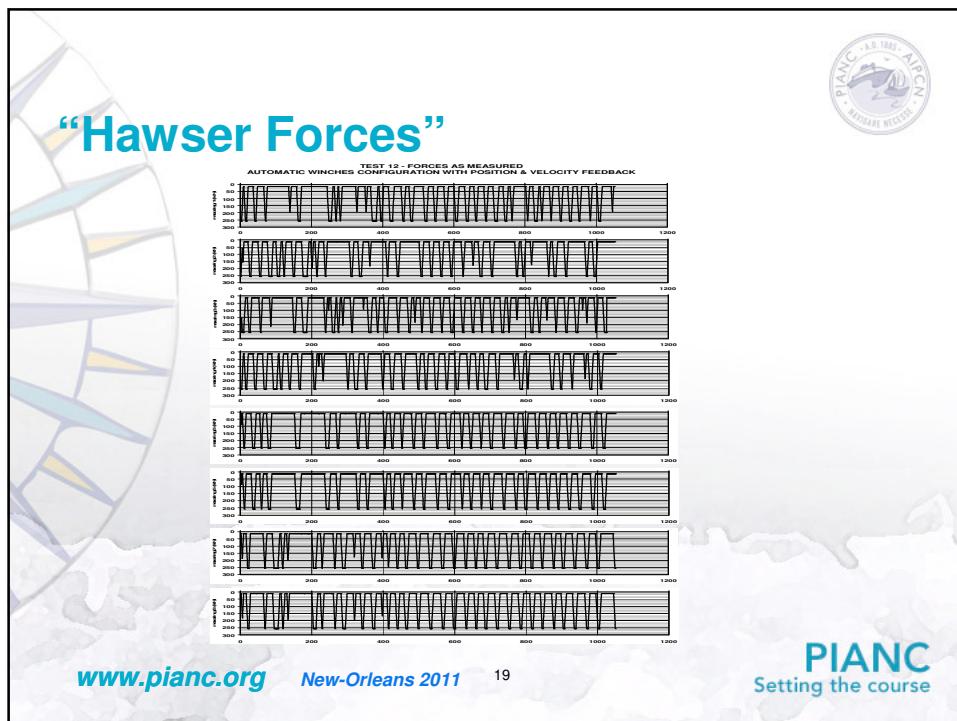
Control algorithm:

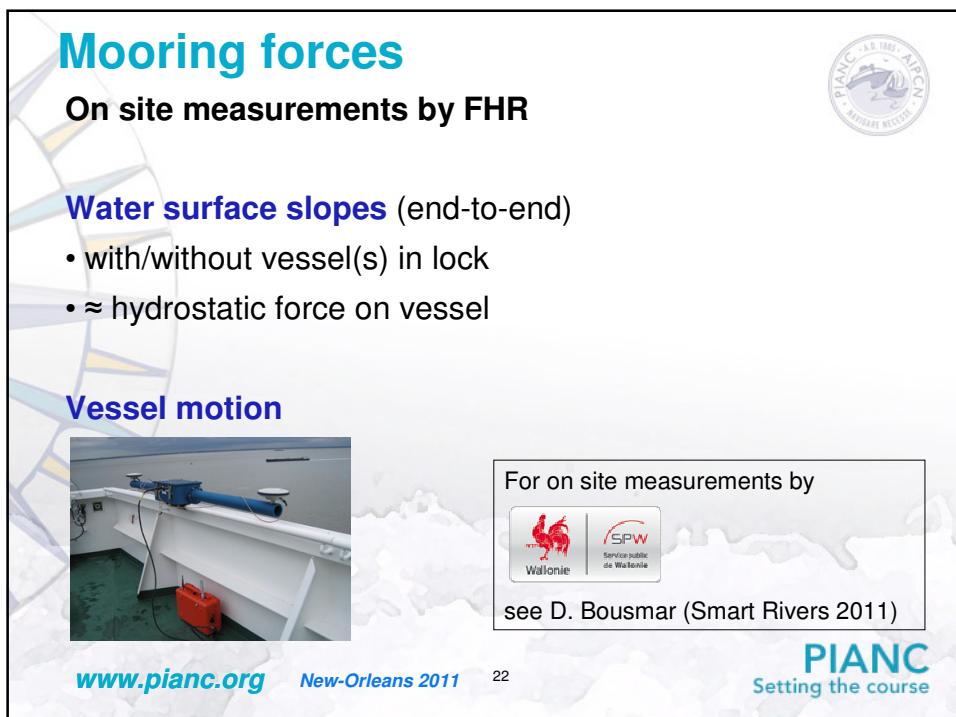
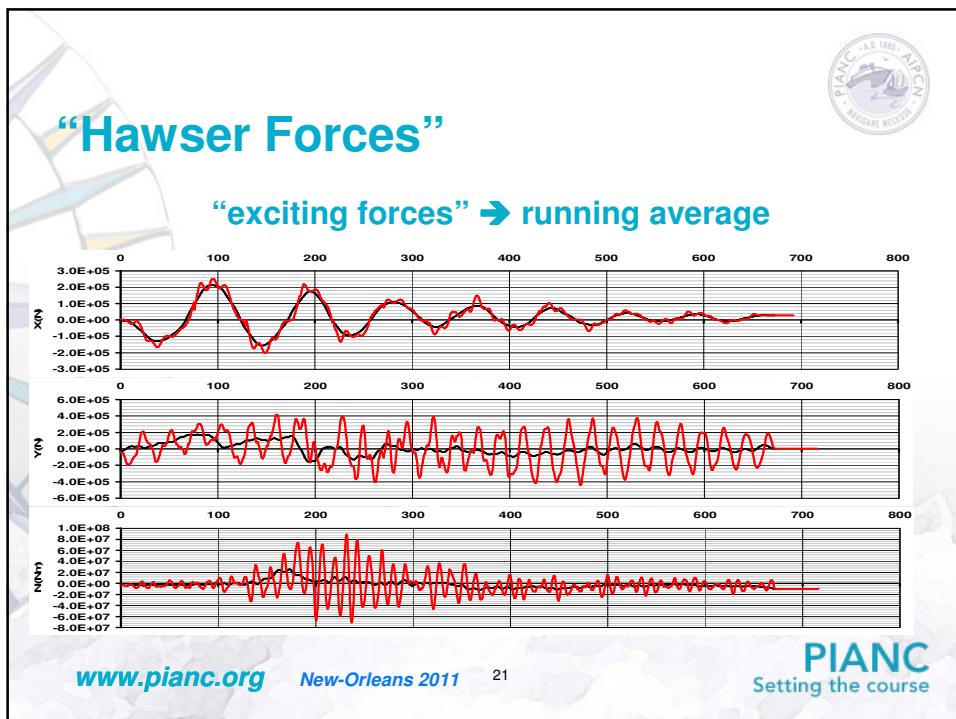
- Initially constant force of 0.2 MBS to each line.
- If bow/stern is closer to SB side of the lock, or if bow/stern is moving to SB, breast line fore/aft starboard will be rendered.
- If bow/stern is closer to PS of lock, or if bow/stern is moving to port, breast line fore/aft port will be rendered.
- If ship is moving ahead/astern, or if ship's position is ahead/astern of her initial position, springs fore/aft will be rendered.
- In case a line is rendered, it is assumed that line force is reduced to 1% of MBS. It is assumed that line length can be adjusted with a maximum speed of 0.25 m/s.
- During the simulations, the necessity of rendering the lines is evaluated with a time interval of 2 s.

“Hawser Forces”

TEST 12 - FORCES AS MEASURED
AUTOMATIC WINCHES CONFIGURATION WITH POSITION & VELOCITY FEEDBACK









More on mooring forces

- **T. De Mulder (2007):** *Specification of hawser force criteria.* Report TO1-Task 1.2.4-CNR-FHR-R013 of Consorcio Pos-Panamax, commissioned by ACP.

➔ Report n°106 of PIANC WG 29 ; CD-ROM ; Directory C14

- **T. De Mulder (2009):** *Mooring forces and ship behaviour in navigation locks.* Paper 7 in Int. Workshop on "Innovations in Navigation Lock Design", PIANC-Brussels, 15-17 Oct. 2009.

➔ <http://www-new.anast.ulg.ac.be/index.php/fr/nouveautés/40/94-pianc-workshop-innovations-in-navigation-lock-designg>

- **T. De Mulder et al. (2010):** *On hawser force criteria for navigation lock design: Case study of maritime locks in Port of Antwerp.* Paper 290 in Proc. PIANC MMX Congress, Liverpool-UK, 10-14 May 2010.



More on mooring forces

- **on site tests & inquiries** at stakeholders
(e.g. systematic variation of valve opening laws)

- **intercomparison** of scale models, numerical models, on site measurements
 - in **absolute** terms (?)
 - in **relative** terms (!): for a given lock (+vessel), do results show the same **trends?**
 - w.r.t. influence on peak force / peak slope of:
 - ➔ initial head
 - ➔ valve opening time
 - ➔ distance bow to filling gate
 - ➔ ship size (displacement, blockage,...)
 - ➔ etc.